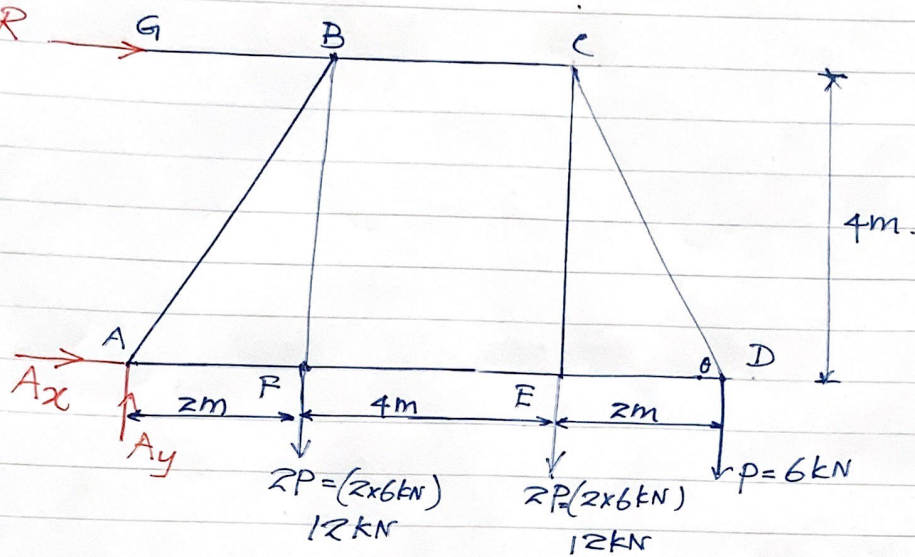


BYG 140 Konstruksjonsmekanikk 1

May 19, 2020

(Q1)(a)



Considering equilibrium,

$$\rightarrow \sum F_x = A_x + R = 0$$

$$\uparrow \sum F_y = A_y - 12 - 12 - 6 = 0$$

$$A_y = 30 \text{ kN} \checkmark$$

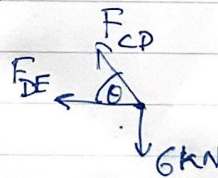
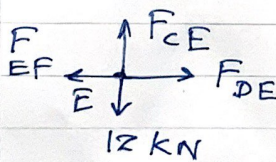
$\curvearrowleft M_A$

$$= 4R + (12 \times 2) + (12 \times 6) + (6 \times 8) = 0$$

$$R = -36 \text{ kN} \checkmark$$

$$A_x = -(-36) = 36 \text{ kN} \checkmark$$

(b) Using method of joints.



$$\sin \theta = \frac{4}{\sqrt{16+4}} = \frac{4}{\sqrt{20}} = \frac{2}{\sqrt{5}}$$

$$\cos \theta = \frac{2}{\sqrt{20}} = \frac{2}{2\sqrt{5}}$$

$$F_{CE} = 12 \text{ kN (T)} \checkmark$$

$$\uparrow F_{CD} \sin \theta - 6 = 0$$

$$F_{CD} = \frac{6}{\frac{2}{\sqrt{20}}} = \frac{6 \times \sqrt{20}}{2} = 6\sqrt{5} \text{ kN}$$

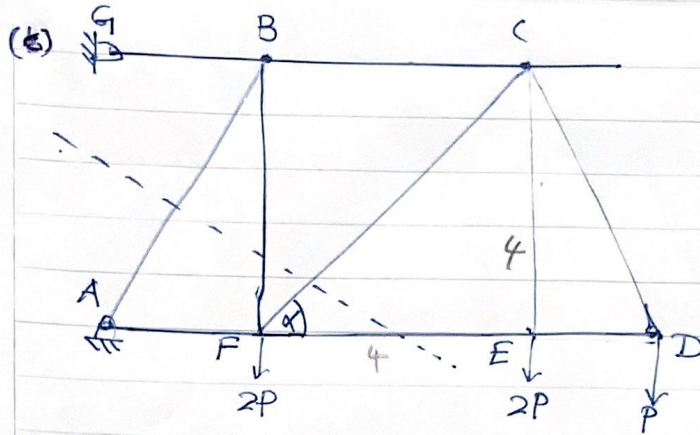
$$F_{EF} = F_{DE} = 3 \text{ kN (C)} \checkmark$$

$$\leftarrow F_{DE} + F_{CD} \cos \theta = 0$$

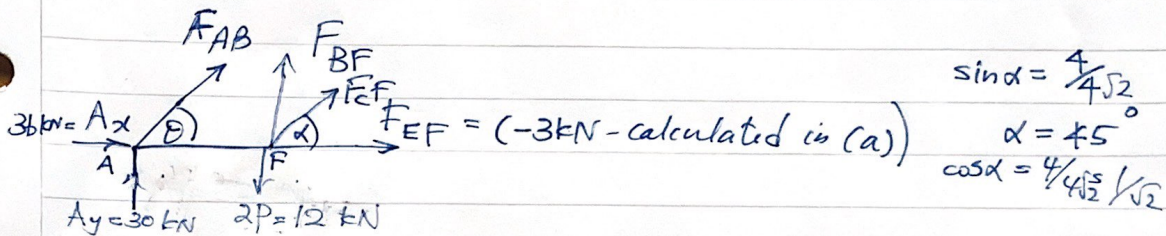
$$F_{DE} = - \frac{6 \times \sqrt{20}}{2} \times \frac{2}{2\sqrt{5}}$$

$$F_{DE} = -3 \text{ kN}$$

$$F_{DE} = 3 \text{ kN (C)} \checkmark$$



Using method of section.



$$\sin \alpha = \frac{4}{4\sqrt{2}}$$

$$\alpha = 45^\circ$$

$$\cos \alpha = \frac{4}{4\sqrt{2}} \cdot \frac{1}{\sqrt{2}}$$

$$\rightarrow \sum F_x = F_{AB} \cos \theta + A_x + F_{EF} + F_{CF} \cos \alpha = 0$$

$$\sqrt{F} = -F_{AB} \sin \theta \times 2 - (30 \times 2) = 0 \rightarrow 15\sqrt{5}$$

$$F_{AB} = \frac{-30}{\frac{1}{\sqrt{2}}} = -33.54 \text{ kN} = \underline{33.54 \text{ kN (e)}}$$

$$F_{CF} \left(\frac{1}{\sqrt{2}} \right) = -(-33.54) \cdot \frac{2}{15\sqrt{5}} - 36 - (-3) = -18 \text{ kN} = \underline{18\sqrt{2} \text{ kN (e)}}$$

(25.46 kN)

$$\uparrow \sum F_y = A_y + F_{AB} \sin \theta + F_{BF} + F_{CF} \sin \alpha - 12 = 0$$

$$F_{BF} = -30 - (-15\sqrt{5} \cdot \frac{2}{\sqrt{5}}) - (-18\sqrt{2} \cdot \frac{1}{\sqrt{2}}) + 12$$

$$F_{BF} = 18 + 12$$

$$F_{BF} = \underline{30 \text{ kN (T)}}$$

(d) cross sectional area of CD

$$F_{CD} = 6.71 \text{ kN}$$

$$\sigma_{all} \geq \frac{F_{CD}}{\text{area}} \Rightarrow \text{area} \geq \frac{6.71 \times 10^3 \text{ N}}{150 \text{ N/mm}^2} = \underline{44.73 \text{ mm}^2}$$

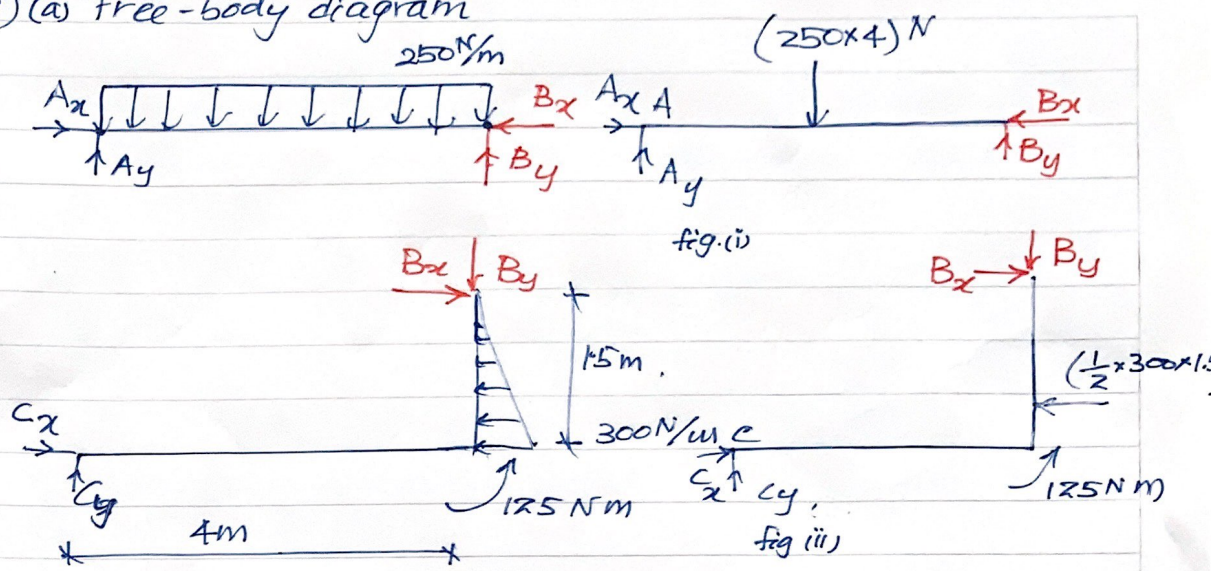
(e) change in length of CD

$$\frac{F}{A} = E \frac{\Delta}{L} \quad \Delta = \frac{FL}{AE} = \frac{(6.71 \times 10^3) \text{ N} \cdot 2\sqrt{5} \times 10^3}{44.73 \text{ mm}^2 \times 205 \times 10^3 \text{ N/mm}^2} = 3.27 \text{ mm}$$

∴ change in length is not equal to the vertical displacement of point D.

- (f) If the self-weight of the truss members are significantly large,
- we have to consider member weights into consideration.
 - we distribute member weights to joint as a joint force.
 - Then calculate member force using any method.

(Q2) (a) free-body diagram



(b) Support reactions

taking moment at A → fig (i)

$$\sum M_A = 4B_y - (250 \times 4 \times 2) = 0 \quad B_y = \underline{500 \text{ N}}$$

taking moment at C (Fig ii)

$$\begin{aligned} \sum M_C &= -4B_y - 1.5B_x + \left(\frac{1}{2} \times 300 \times 1.5\right) \cdot \frac{1.5}{3} + 125 = 0 \\ +1.5B_x &= \left(\frac{1}{2} \times 300 \times 1.5 \times \frac{1.5}{3}\right) + 125 - 4 \times 500 \\ B_x &= \underline{\underline{-1175 \text{ N}}} \end{aligned}$$

Fig (i)

$$\begin{aligned} \rightarrow A_x + B_x &= 0 \quad A_x = -(-1175) = \underline{1175 \text{ N}} \\ \uparrow A_y + B_y - 1000 &= 0 \quad A_y = 1000 - 500 = \underline{500 \text{ N}} \end{aligned}$$

Fig (ii)

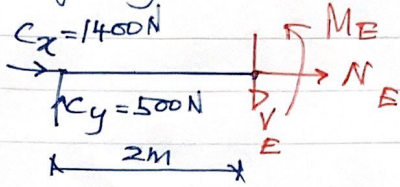
$$\uparrow C_y - B_y = 0 \quad C_y = \underline{500 \text{ N}}$$

$$\begin{aligned} \rightarrow C_x + B_x - \left(\frac{1}{2} \times 300 \times 1.5\right) &= 0 \quad C_x = \left(\frac{1}{2} \times 300 \times 1.5\right) - (-1175) \\ C_x &= \underline{1400 \text{ N}} \end{aligned}$$

(c) Internal forces

imaginary cut at E

considering equilibrium



$$\uparrow C_y - V_E = 0$$

$$V_E = \underline{500 \text{ N}}$$

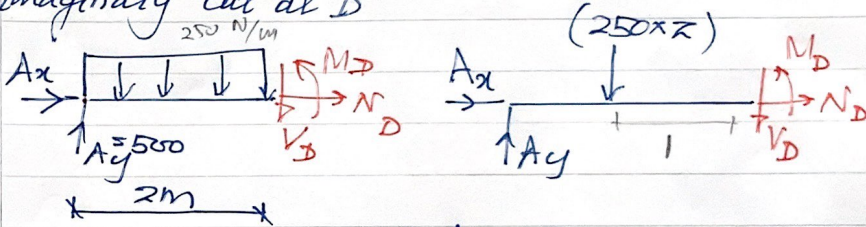
$$\rightarrow N_E + C_x = 0$$

$$N_E = \underline{-1400 \text{ N}}$$

$$\curvearrowright M_E - 500 \text{ N} \cdot 2 = 0$$

$$M_E = \underline{1000 \text{ Nm}}$$

imaginary cut at D



$$\rightarrow A_x + N_D = 0$$

$$N_D = \underline{+1175 \text{ N}}$$

$$\uparrow A_y - V_D - 500 = 0$$

$$-V_D = 500 - 500 = 0$$

$$V_D = \underline{0}$$

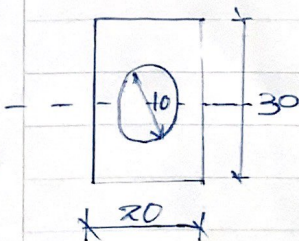
$$\curvearrowright M_D + (250 \times 2 \times 1) - 500 \times 2 = 0$$

$$M_D = 1000 - 500 = \underline{500 \text{ Nm}}$$

(d) cross sectional area

$$= (30 \times 20) - (\pi \times 5^2) = \underline{521.46 \text{ mm}^2}$$

moment of inertia



$$I = \left(\frac{1}{12} \cdot 20 \cdot 30^3 \right) - \left(\frac{1}{4} \pi 5^4 \right)$$

$$= \underline{44509.13 \text{ mm}^4}$$

(e) maximum normal stress

$$\sigma = \text{Normal stress due to axial force } (\sigma_N) + \text{Normal stress due to bending } (\sigma_M)$$

$$\sigma = \frac{P}{A} + \left(\frac{-MY}{I} \right)$$

at D

$$\frac{1175 \text{ N}}{521.46 \text{ mm}^2} + \left(\frac{500 \times 10^3 \times 15}{44509.13} \right) = \underline{\underline{170.758 \text{ MPa}}}$$

at E

$$= \frac{-1400 \text{ N}}{521.46 \text{ mm}^2} + \left(\frac{-1000 \times 10^3 \text{ Nmm} \times 15 \text{ mm}}{44509.13 \text{ mm}^4} \right) = \underline{\underline{-339.69 \text{ MPa}}}$$

Maximum shear stress

at D

$$\tau_{\max} = \frac{VQ}{It} = \underline{\underline{0}}$$

at E

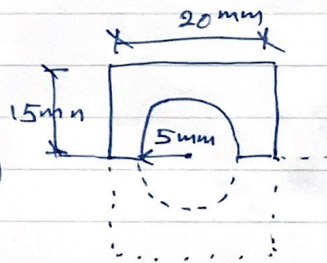
$$V = 500 \text{ N}$$

$$t = 10 \text{ mm}$$

$$I = 44509.13 \text{ mm}^4$$

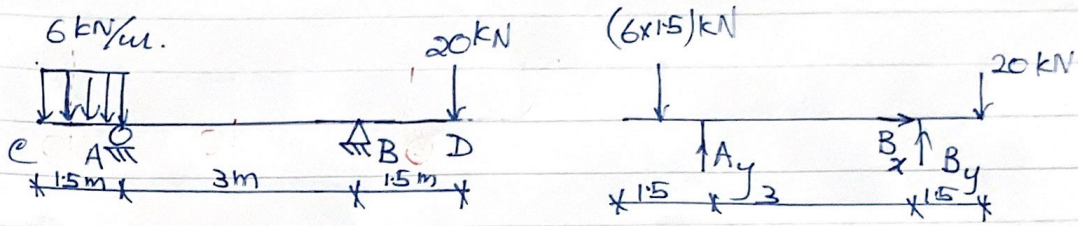
$$Q = \sum A \bar{y}^2 = (20 \times 15) \cdot 15 - \left(\frac{\pi \times 5^2}{2} \cdot \frac{4 \times 5}{3\pi} \right)$$

$$= 2166.667 \text{ mm}^3$$



$$\tau_{\max} = \frac{500 \text{ N} \times 2166.67 \text{ mm}^3}{44509.13 \text{ mm}^4 \times 10 \text{ mm}} = \underline{\underline{2.43 \text{ MPa}}}$$

(Q 3) (a)



considering equilibrium.

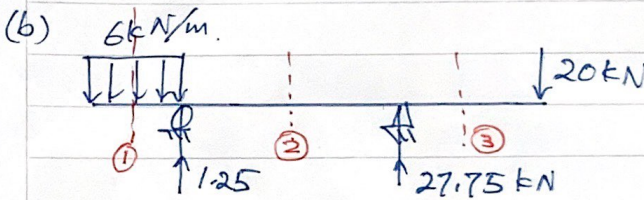
$\rightarrow B_x = 0$

$\uparrow A_y + B_y = 20 + (6 \times 3) = 29$

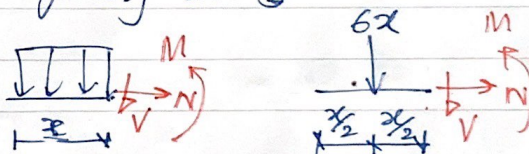
$\sum \bar{B} (-20 \times 1.5) - (A_y \cdot 3) + (9 \times 3 + \frac{15}{2}) = 0$

$A_y = 1.25 \text{ kN}$

$B_y = 29 - 1.25 = 27.75 \text{ kN}$



imaginary cut ①



| x | V | M |
|-----|----|-------|
| 0 | 0 | 0 |
| 1.5 | -9 | -6.75 |

considering equilibrium

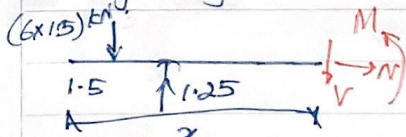
$\downarrow V + 6x = 0$

$V = -6x$

$\curvearrowright M + 6x \cdot \frac{x}{2} = 0$

$\frac{1}{2} M = -3x^2$

imaginary cut ②



| x | V | M |
|-----|-------|-------|
| 1.5 | -7.75 | -6.75 |
| 4.5 | -7.75 | -30 |

$\downarrow V + (6 \times 1.5) - 1.25 = 0$

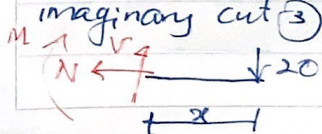
$V = -7.75$

$\curvearrowright M + (9 \times (\frac{x-1.5}{2})) - 1.25(x-1.5) = 0$

$M + 9x - 6.75 - 1.25x + 1.875 = 0$

$M = -7.75x + 4.875$

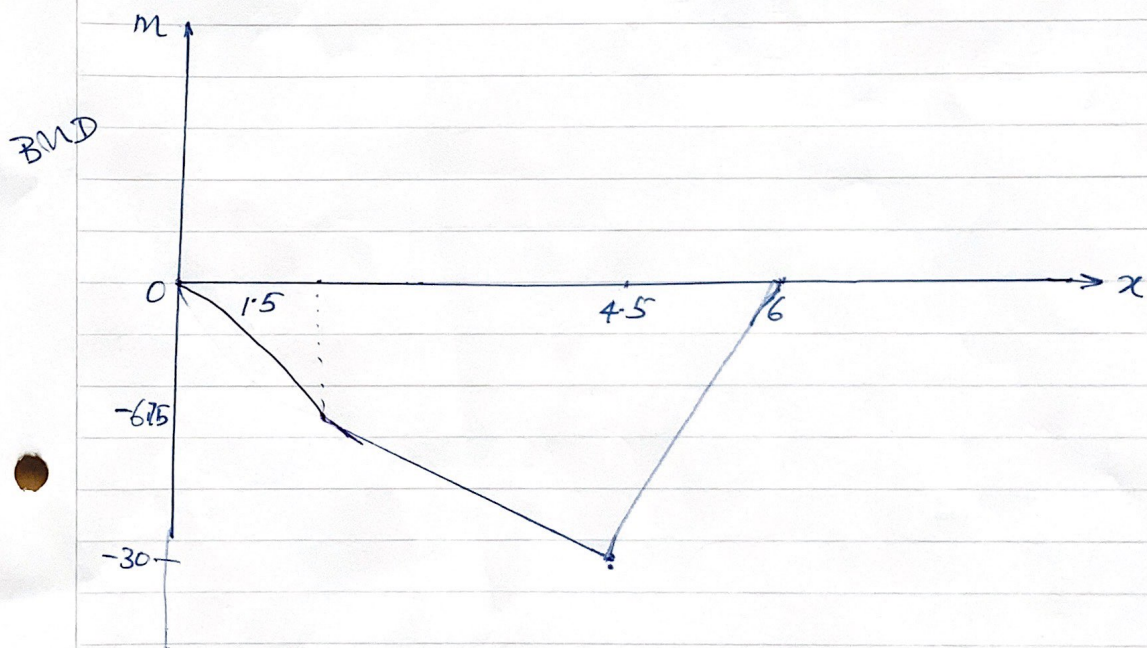
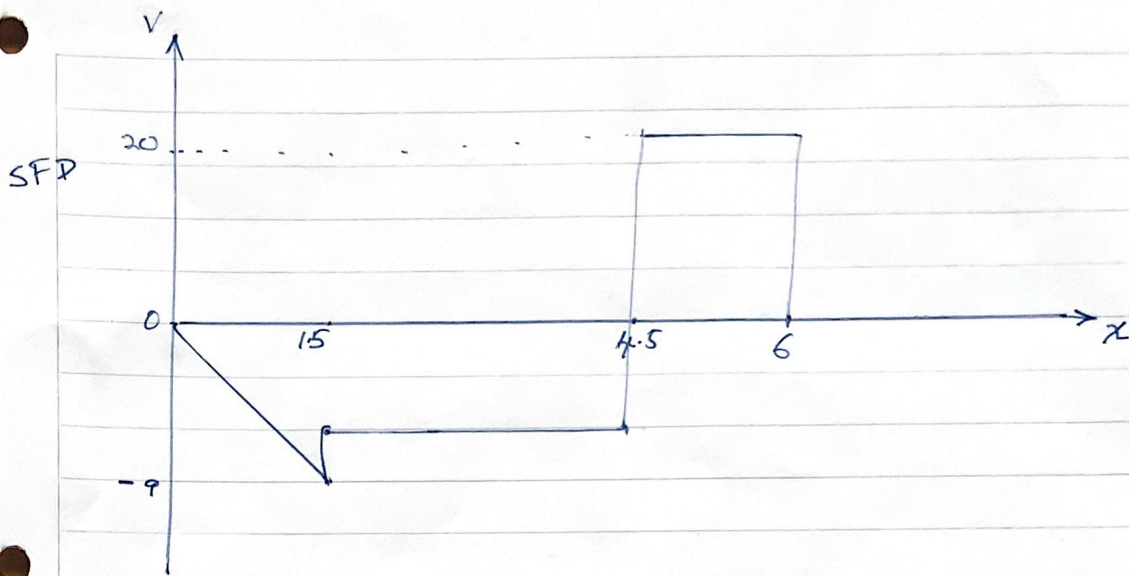
imaginary cut ③



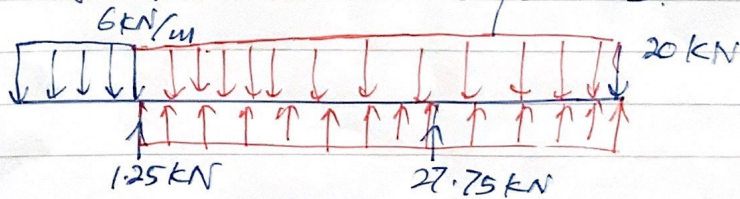
$\uparrow V - 20 = 0 \quad V = 20$

$\curvearrowright M + 20x = 0 \quad M = 20x$

| x | V | M |
|-----|----|----|
| 0 | 20 | 0 |
| 1.5 | 20 | 30 |



(d) Deflection at the end point D.



$$M = 1.25 \langle x - 1.5 \rangle^1 - \frac{6}{2} \langle x - 0 \rangle^2 + \frac{6}{2} \langle x - 1.5 \rangle^2 + 27.75 \langle x - 4.5 \rangle^1$$

$$EI \frac{d^2v}{dx^2} = M(x)$$

$$EI \frac{d^2v}{dx^2} = 1.25 \langle x-1.5 \rangle^1 - 3x^2 + 3 \langle x-1.5 \rangle^2 + 27.75 \langle x-4.5 \rangle^1$$

$$EI \frac{dv}{dx} = \frac{1.25}{2} \langle x-1.5 \rangle^2 - x^3 + \frac{3}{3} \langle x-1.5 \rangle^3 + \frac{27.75}{2} \langle x-4.5 \rangle^2 + C_1$$

$$EI v = \frac{1.25}{6} \langle x-1.5 \rangle^3 - \frac{x^4}{4} + \frac{3}{12} \langle x-1.5 \rangle^4 + \frac{27.75}{6} \langle x-4.5 \rangle^3 + C_1 x + C_2$$

Using boundary conditions.

$$x=1.5 \quad v=0$$

$$x=4.5 \quad v=0$$

from eq. (2)

$$EI \cdot 0 = -\frac{1.5^4}{4} + C_1 \cdot 1.5 + C_2 \quad C_1 \cdot 1.5 + C_2 = 1.266 \quad \text{--- (1)}$$

$$0 = \frac{1.25}{6} (4.5-1.5)^3 - \frac{4.5^4}{4} + \frac{3}{12} (4.5-1.5)^4 + 4.5 C_1 + C_2$$

$$4.5 C_1 + C_2 = 76.64 \quad \text{--- (2)}$$

$$\text{①} \times \text{②} \Rightarrow C_1 = 25.12$$

$$C_2 = -36.4$$

Then

$$v = \frac{1}{EI} \left\{ \frac{1.25}{6} \langle x-1.5 \rangle^3 - \frac{x^4}{4} + \frac{3}{12} \langle x-1.5 \rangle^4 + \frac{27.75}{6} \langle x-4.5 \rangle^3 + 25.12x - 36.4 \right\}$$

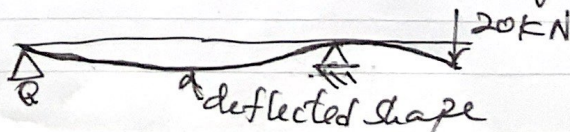
$$\text{At D, } x=6\text{m}$$

Deflection

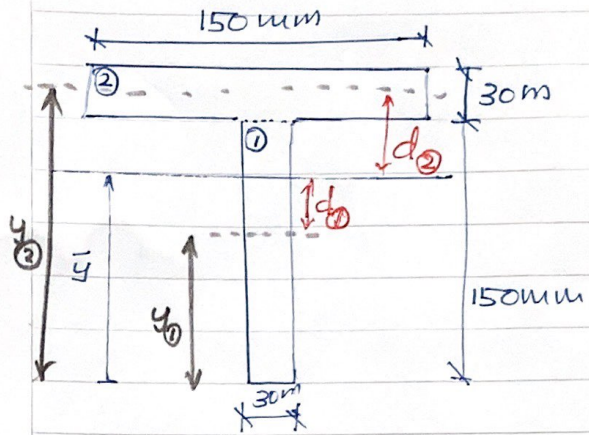
$$v = \frac{1}{EI} \left\{ \frac{1.25}{6} (6-1.5)^3 - \frac{6^4}{4} + \frac{3}{12} \langle 6-1.5 \rangle^4 + \frac{27.75}{6} (6-4.5)^3 + (25.12 \times 6) - 36.4 \right\}$$

$$v = \frac{-72.570625}{210 \times 10^9 \times 2.7 \times 10^{-3}} = 0.0128\text{m} \quad v = \underline{\underline{12.8\text{mm}}}$$

(e) magnitude of the deflection at D will reduce. Due to the change of deflected shape



(c) moment of inertia.



$$\bar{y} = \frac{A_1 \tilde{y}_1 + A_2 \tilde{y}_2}{A_1 + A_2}$$

$$\bar{y} = \frac{(150 \times 30 \times \frac{150}{2}) + (150 \times 30 \times (\frac{150+30}{2}))}{(150 \times 30) + (150 \times 30)}$$

$$= 120 \text{ mm}$$

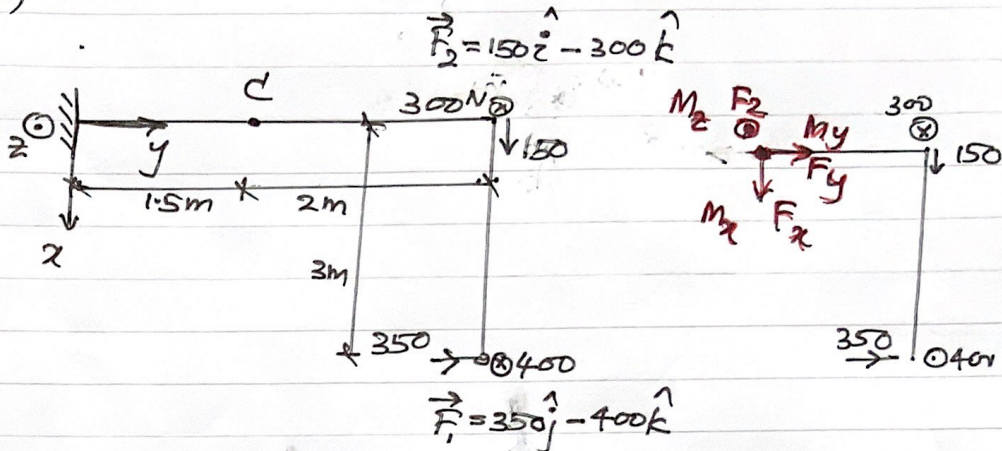
$$I = \left[\left(\frac{1}{12} \cdot 30 \times 150^3 \right) + (150 \times 30) \times (120 - 75)^2 \right] +$$

$$\left[\left(\frac{1}{12} \cdot 150 \times 30^3 \right) + (150 \times 30) \times (165 - 120)^2 \right]$$

$$= \underline{27000000 \text{ mm}^4}$$

(Q4)

(a)



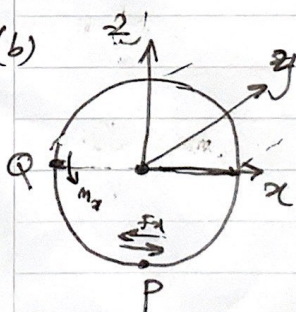
$$\begin{aligned} \sum F_x = F_x + 150 &= 0 & F_x &= -150 \text{ N} \\ \sum F_y = F_y + 350 &= 0 & F_y &= -350 \text{ N} \\ \sum F_z = F_z - 300 - 400 &= 0 & F_z &= 700 \text{ N} \end{aligned}$$

$$\sum M_x = M_x - (300 \times 2) - (400 \times 2) = 0 \quad M_x = 1400 \text{ Nm}$$

$$\sum M_y = M_y + (400 \times 3) = 0 \quad M_y = -1200 \text{ Nm}$$

$$\sum M_z = M_z + (350 \times 3) - (150 \times 2) = 0 \quad M_z = -750 \text{ Nm}$$

(b)



$$\sigma_p = \frac{F_y}{A} - \frac{M_z}{I} r$$

$$= \frac{350}{\pi r^2} - \frac{1400 \times 10^{-6}}{\frac{1}{4} \pi r^4} r = \frac{350}{\pi r^2} - \frac{1400 \times 10^{-3}}{\frac{\pi r^3}{4}}$$

diameter (d) = 60 mm

$$\sigma_p = -65.866 \text{ N/mm}^2$$

$$\tau_p = \frac{T r}{J} - \frac{|F_x| Q}{I(z)}$$

$$= \frac{1200 \times 0.03}{1.2723 \times 10^{-6}} - \frac{150 \times 1.8 \times 10^{-3}}{6.3617 \times 10^{-6} \times 0.06}$$

$$= 28.22 \text{ N/mm}^2$$

$$J = \frac{\pi}{2} r^4 = 1.272 \times 10^{-6} \text{ m}^4$$

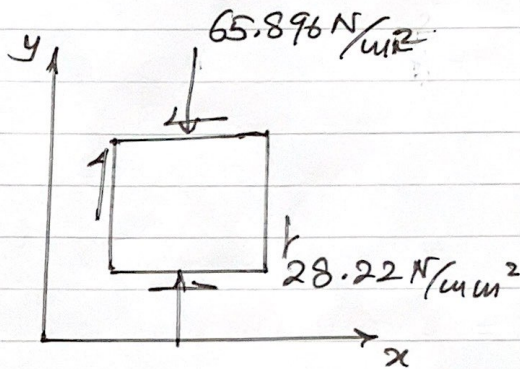
$$I = \frac{\pi}{4} r^4 = 6.36 \times 10^{-7} \text{ m}^4$$

$$Q = \frac{\pi r^3}{2} \cdot \frac{4r}{3\pi} = 1.8 \times 10^{-5} \text{ m}^3$$

$$\sigma_z = \frac{Fy}{A} - \frac{M_z I}{J} = \frac{350}{\pi r^2} - \frac{750 \times 10^3 r}{\frac{1}{4} \pi r^4} = \underline{\underline{-35.294 \text{ N/mm}^2}}$$

$$\tau_z = \frac{Tr}{J} - \frac{FR_2 I Q}{I (2r)} = \underline{\underline{27.96 \text{ N/mm}^2}}$$

At P



$$\sigma_x = 0$$

$$\sigma_y = -65.896 \text{ N/mm}^2$$

$$\tau_{xy} = -28.22 \text{ N/mm}^2$$

$$(c) \sigma_{1,2} = \frac{\sigma_1 + \sigma_2}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau^2}$$

$$\sigma_{1,2} = \frac{-65.896}{2} \pm \sqrt{\left(\frac{65.896}{2}\right)^2 + (28.22)^2}$$

$$\sigma_1 = \underline{\underline{10.43 \text{ N/mm}^2}}$$

$$\sigma_2 = \underline{\underline{-76.33 \text{ N/mm}^2}}$$

$$2\theta_p = \tan^{-1}\left(\frac{\tau}{\left(\frac{\sigma_x - \sigma_y}{2}\right)}\right) = \tan^{-1}\left(\frac{28.22}{\left(\frac{65.896}{2}\right)}\right) = 40.58$$

$$\theta_p = \underline{\underline{20.29^\circ}}$$

(d) Unstable structure. It becomes a mechanism.